

Hydraulics

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Char-Lynn®
Disc Valve Hydraulic Motors

11-01-878
EN-0201



**6000 Series
Hydraulic Motors**

We Manufacture

Solutions

Catalog of Disc Valve Hydraulic Motors from One of the World's Leading Manufacturers of Off Highway Mobile Components — Eaton Hydraulics

In the late 1950's the original low speed, high torque hydraulic motor was developed from a pump gerotor element consisting of an internal gear ring and a mating gear or star. While attaching the internal gear ring to the housing as a non moving part, oil was ported to pressurize and turn the internal star in an orbit around a center point. This slow turning star coupled with a splined drive to the output shaft became the Char-Lynn® Orbit® motor. A few years after this original Char-Lynn Orbit motor was introduced another original motor concept went into production. This motor had rolls incorporated into the internal gear ring, this element was identified by the name Geroler® and is a registered trade name of Eaton Hydraulics. From these early years the Geroler motor has seen many design changes

to make these Geroler motors the best the industry has to offer. Examine the simplicity of these Geroler disc valve motors shown below. Also examine all the following pages for high value Char-Lynn disc valve motors from Eaton Hydraulics.

Geroler® Displacement Mechanism

Motors with the Geroler element provide high starting and running torque. The Geroler element minimizes friction and thereby increases efficiency while providing smooth output shaft rotation even at very low speeds. Motor shaft rotation can be instantly reversed by changing direction of input/output flow while generating equal torque in either

direction. The displacements available provide a wide variety of speeds and torques from any Series motor.

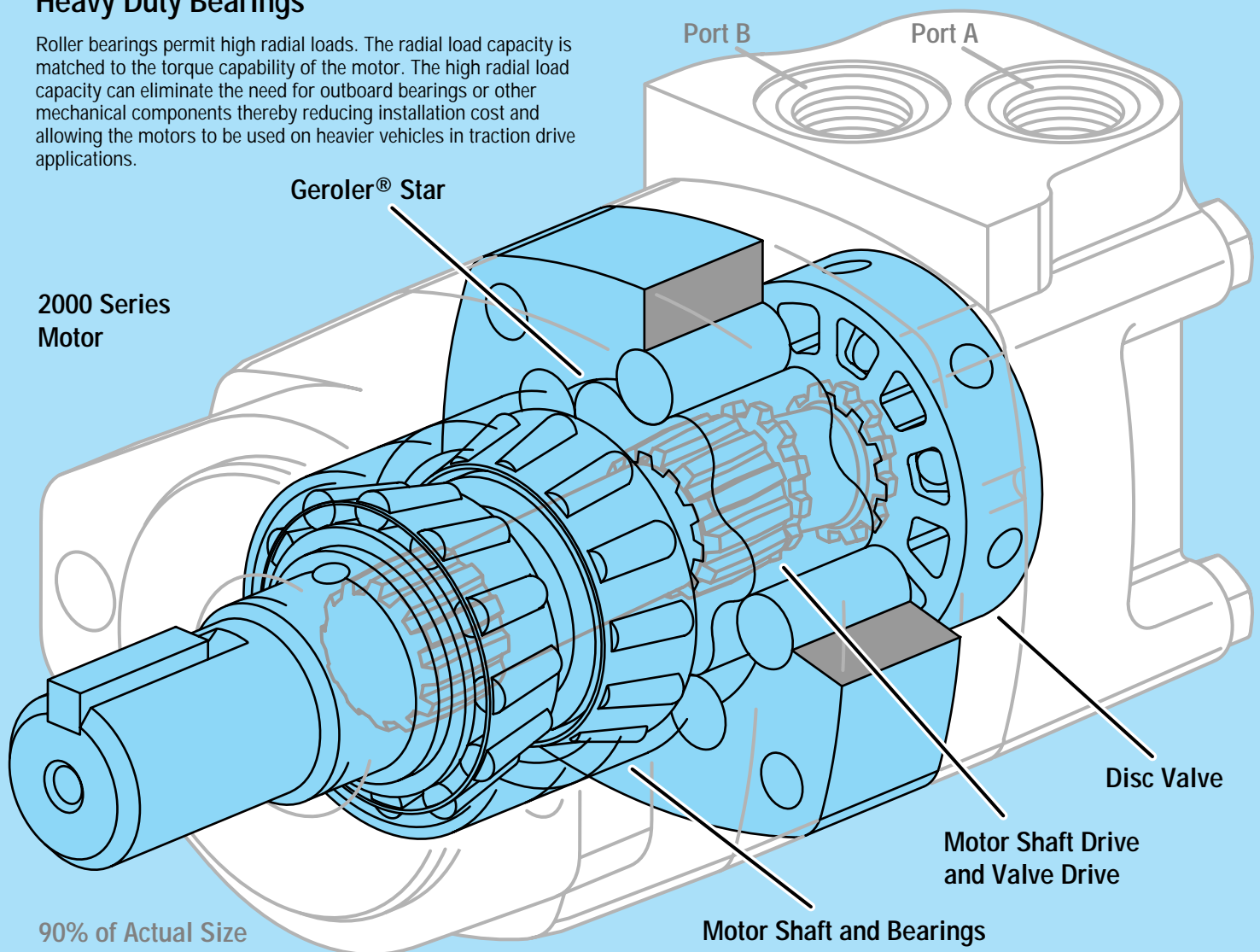
Disc Valve

The function of the disc valve is to distribute fluid to the Geroler pockets. The pressure balanced sealing surface on the valve face maintains minimal leakage. Char-Lynn disc valve motors can be used in the same system with a radial piston pump and also in closed loop systems. The patented wear compensated disc valve provides top performance.

Heavy Duty Bearings

Roller bearings permit high radial loads. The radial load capacity is matched to the torque capability of the motor. The high radial load capacity can eliminate the need for outboard bearings or other mechanical components thereby reducing installation cost and allowing the motors to be used on heavier vehicles in traction drive applications.

2000 Series Motor



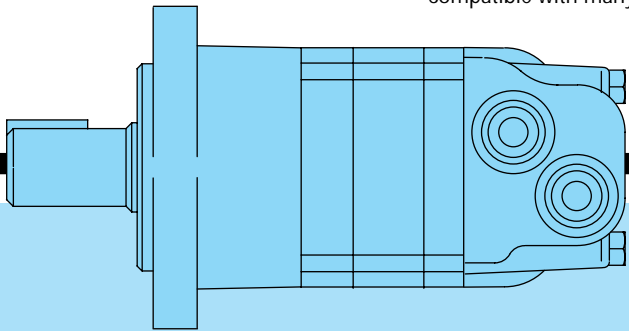
Design Features

Char-Lynn Hydraulic motors provide design flexibility. All disc valve motors are available with various configurations consisting of:

- Displacement (Geroler size)
- Output Shaft
- No Shaft and Bearing Assembly (Bearingless Motor)
- Port Configuration
- Mounting Flange
- Other Special Features

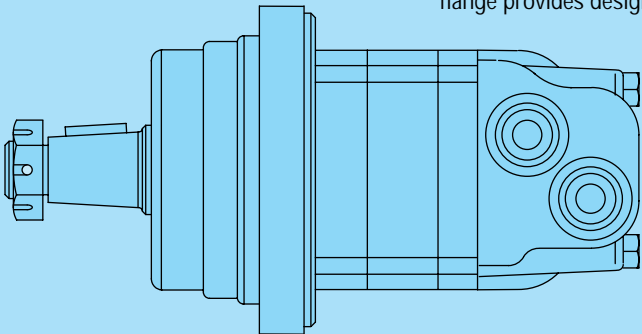
Standard Motor

The standard motor mounting flange is located as close to the output shaft as possible. This type of mounting supports the motor close to the shaft load. This mounting flange is also compatible with many standard gear boxes.



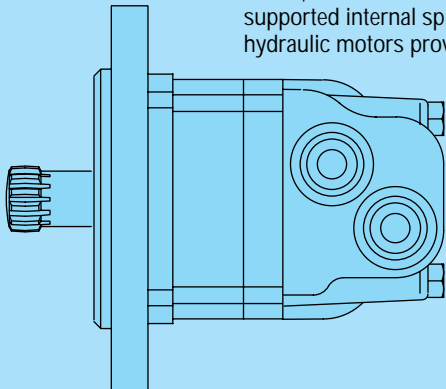
Wheel Motor

The wheel motor mounting flange is located near the center of the motor which permits part or all of the motor to be located inside the wheel or roller hub. In traction drive applications, loads can be positioned over the motor bearings for best bearing life. This wheel motor mounting flange provides design flexibility in many applications.

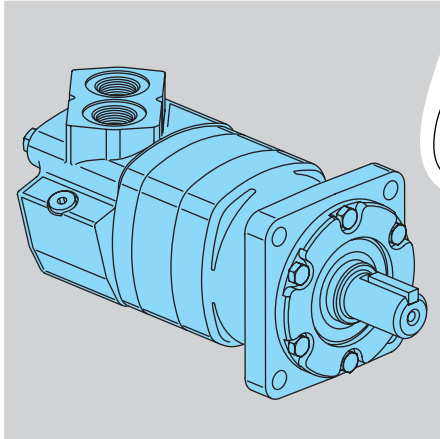


Bearingless Motor

This bearingless motor has the same drive components as the standard and wheel motors (with the exception that the motor is assembled without the output shaft, bearings and bearing housing). The bearingless motor is especially suited for applications such as gear boxes, winch drives, reel and roll drives. Bearingless motor applications must be designed with a bearing supported internal spline to mate with the bearingless motor drive. Product designs using these hydraulic motors provide considerable cost savings.



6000 Series



6000

6000 Series

Geroler® Element	9 Displacements
Flow LPM [GPM]	150 [40] Continuous**
	225 [60] Intermittent*
Speed	Up to 866 RPM
Pressure Bar [PSI] ...	200 [3000] Cont.
	300 [4500] Inter.
Torque Nm [lb-in]....	1685 [14920] Cont.
	1875 [16580] Inter.

6000 Series Displacement Size = cubic centimeter per shaft revolution (cm^3/r)
= cubic inch per shaft revolution ([in^3/r])

- 195 [11.9]
- 245 [15.0]
- 310 [19.0]
- 390 [23.9]
- 490 [30.0]
- 625 [38.0]
- 740 [45.0]**
- 805 [49.0]**
- 985 [60.0]

Mounting Flange

- 4 Bolt (Bearingless) 127,0 [5.00] Pilot Dia. and 14,3 [.56] Dia. Mounting Holes on 161,9 [6.38] Dia. B.C.
- 4 Bolt (SAE CC)(Standard) 127,0 [5.00] Pilot Dia. and 14,3 [.56] Mounting Holes on 161,9 [6.38] Dia. B.C.
- 4 Bolt (Wheel) 139,7 [5.50] Pilot Dia. and 14,3 [.56] Dia. Mounting Holes on 184,1 [7.25] Dia. B.C.

Output Shaft

- Bearingless
- 1-1/2 inch Dia. Straight with Straight Key, 3/8-16 Threaded Hole and 56,7[2.23] Max. Coupling Length
- 1-3/4 inch Dia. Tapered with Straight Key and 1-1/4—18 UNEF Slotted Hex. Nut
- 1-1/2 inch Dia. Splined 17 T with 40,3 [1.59] Min. Full Spline Length and 3/8-16 Threaded Hole
- 40 mm Dia. Straight with Straight Key, M12 x 1,75-6H Threaded Hole

Port Type

- 1-5/16-12 O-ring with 7/16-20 O-ring Case Drain and Shuttle Valve
- G 1 (BSP) O-ring with G 1/4 (BSP) O-ring Case Drain and Check Valve
- 3/4 inch 4 Bolt Split Flange with 7/16-20 O-ring Case Drain and Check Valve

Special Features

- Viton Shaft Seal
- Viton Seals
- Hot Oil Shuttle
- Corrosion Protected

*** For performance and dimension data contact your Eaton Hydraulics representative.

** Continuous— (Cont.) Continuous rating, motor may be run continuously at these ratings.

* Intermittent— (Inter.) Intermittent operation, 10% of every minute.

Design Flexibility

Char-Lynn motors are truly built for high torque low speed. A lot of power is derived from this small package. This power advantage provides the designer with a product that can be used for overall compactness in addition to taking full advantage of the high pressure ratings typical of present day hydraulic components.

Char-Lynn Disc Valve hydraulic motors allow the designer to put the power where it is needed. Indeed, these motors are small in size, big in output power. Hence, the small package can eliminate a lot of installation problems. Furthermore, the motors can be mounted directly on the driven device away from the original power source which eliminates other mechanical linkages such as chains, sprockets, belts, pulleys, gears, rotating drive shafts, and universal joints. Several motors can be driven from the same power source and can be connected in series or parallel to each other.

Durability

The design and method of manufacture of three critical drive train components, valve drive, shaft drive, and output shaft, give these motors durability. Consequently, these durable disc valve motors stand up against high hydraulic pressures. Other built in features, such as the rugged **Tapered** roller bearings provide a good match to this tough design.

Performance Rating

Our method of rating these motors recognizes that at slower speeds and flow, higher pressures and torque are permitted. Hence, our performance data shows the complete flow range (down to 1 liter per minute or 1/4 gallon per minute) and speed range (down to one revolution per minute depending on application).

Controllable Speeds

Char-Lynn motors operate at low speeds that remain very near constant even when load varies. Shaft speed is varied smoothly, easily and economically using simple inexpensive controls. Also, these motors are reversible. Consequently, direction of shaft rotation can be changed instantly with equal output torque in either direction.

Dependable Performance

Highly precise manufacturing of parts and the disc valve's unique wear compensating design provide consistent, dependable performance and long life even under varying conditions.

Reliability

Char-Lynn motors are self contained, with hydraulic fluid providing lubrication. These motors are completely sealed so they can operate safely and reliably in hostile environments such as dust, dirt, steam, water, and heat and provide reliable performance.

High Efficiencies

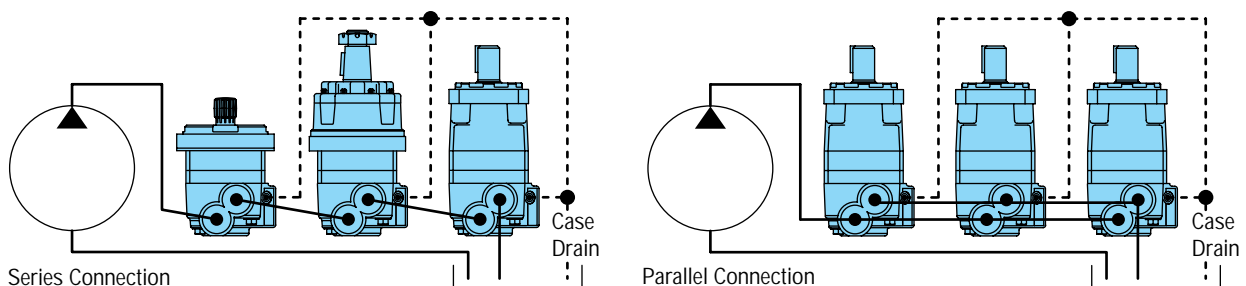
Char-Lynn disc valve motors have high efficiencies providing high output for the pressure and flow supplied. The high mechanical efficiency enables you to obtain a given torque with a smaller displacement motor.

Volumetric efficiency is high and speed is relatively constant with little variation due to changes in load. Speed is controlled easily and smoothly.

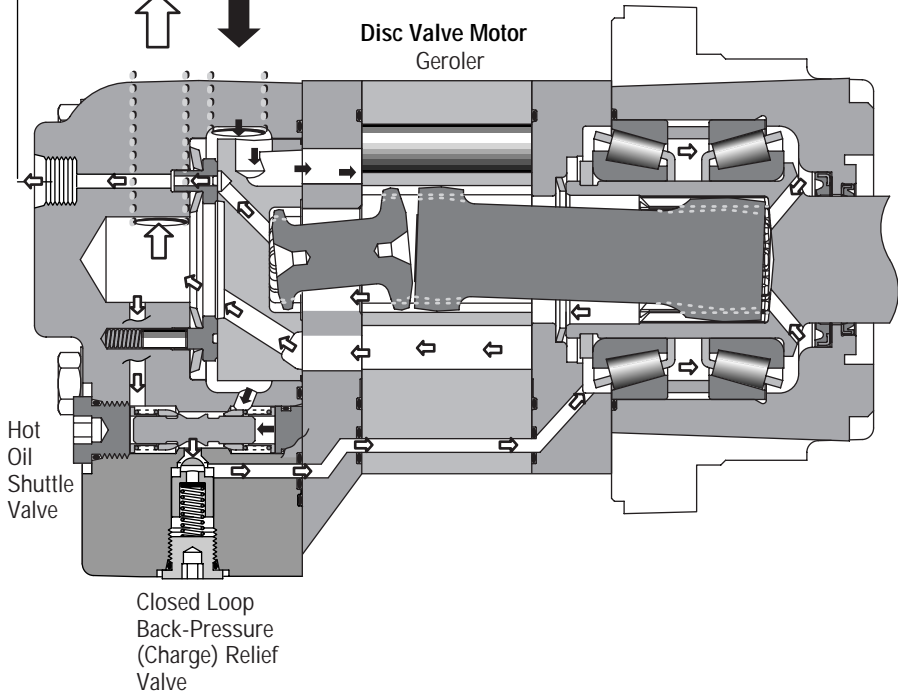
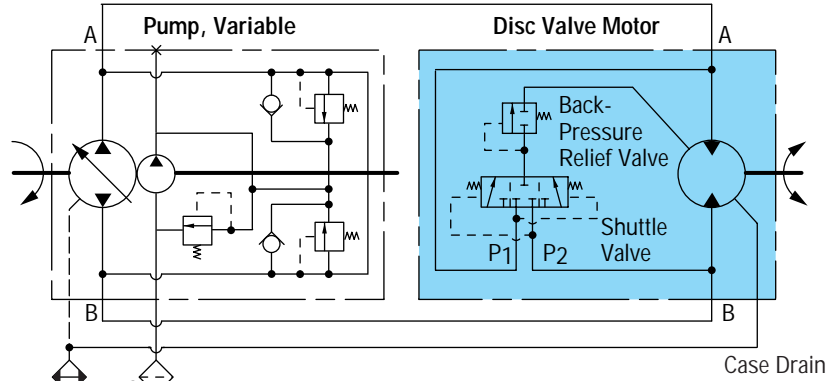
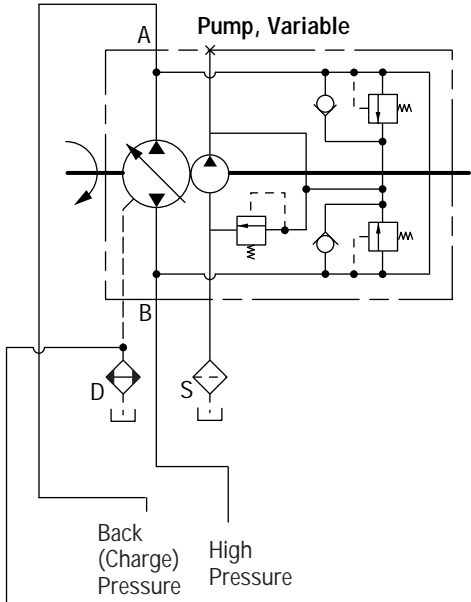
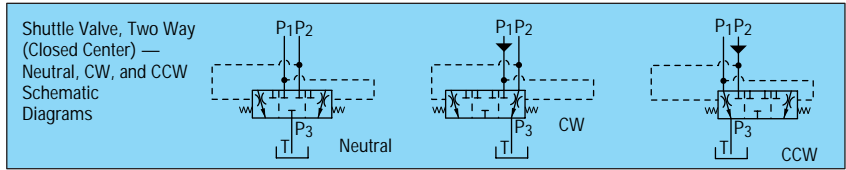
In conclusion, these efficiencies mean less heat buildup in the hydraulic system.

Case Drain and Shuttle Valve Options

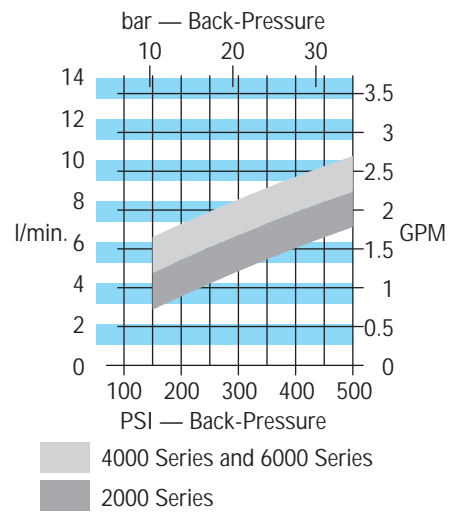
Many hydraulic systems can benefit from the use of a system case drain. Char-Lynn disc valve motors provide this feature built in. One of the advantages for case drain flow is that contamination is flushed from the system. This flushing also aids in cooling the system and lowering the case pressure which will extend motor seal life. With a case drain line in place, oil pressure in the gear box (Bearingless motor applications) can also be controlled. In applications where more system cooling and flushing is required, a shuttle valve option is available in 2000, 4000, and 6000 Series motors.



Typical (Closed Loop) Hydraulic Circuit Shuttle Flow 2000, 4000, 6000 Series



Typical Disc Valve Motor Shuttle Flow with 4,5 bar [65 PSI] Back-Pressure Relief Valve (Typical Data)
Due to Machining Tolerances, Flow May be More or Less



Disc Valve Motor with shuttle valve **must have a case drain to tank**, without this drain line the internal drive splines will not have adequate lubrication.

Low Speed High Torque Hydraulic Motors with Shuttle and Charge Pressure Relief Valve — Patent No. U.S. 4,645,438.

Speed Sensor 2000, 4000 and 6000 Series

Eaton has developed a speed sensor specifically designed for LSHT motors. The design is rugged and fully protected against accidental reverse polarity or short circuit hook up. A built in pull up resistor simplifies installation with control systems.

This sensor is fully compatible with mobile vehicle electrical systems and gives a reliable digital on/off signal over a wide speed range and temperature range. The sensor is field-serviceable; no factory setting or shimming is required.

Supply Voltage: 8 to 24 Vdc (compatible with 12V vehicle systems)

Supply Current: 20 mA max. (Vs) (including internal pull-up resistor)

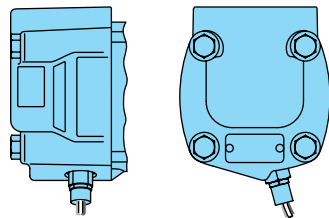
Output Voltage: Low < .5 Vdc @ 10 mA; output is open collector with 10kΩ pull-up resistor.

Connection — standard 3 prong Weatherpack connector with 18 AWG (american wire gage) cables:

- Position A (red) = power supply
- Position B (white) = signal output
- Position C (black) = common

2000 Series

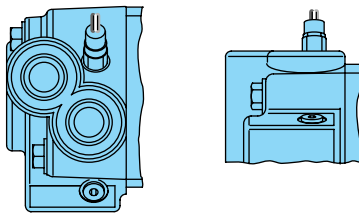
Output Speed Sensor — digital on/off signal from a Hall Effect switch; 30 pulses/revolution



Output Quadrature Speed Sensor — 60 pulses/revolution

4000 Series

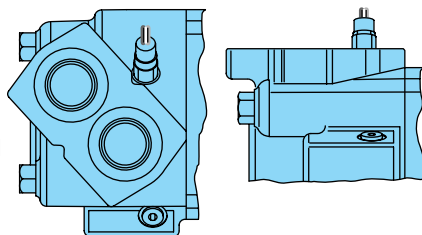
Output Speed Sensor — digital on/off signal from a Hall Effect switch; 36 pulses/revolution



Output Quadrature Speed Sensor — 72 pulses/revolution

6000 Series

Output Speed Sensor — digital on/off signal from a Hall Effect switch; 40 pulses/revolution



Output Quadrature Speed Sensor — 80 pulses/revolution

Quadrature Speed Sensor 2000, 4000 and 6000 Series

Eaton has developed a new **speed** and **direction** sensor, based on the field proven technology of our standard sensor, designed for off road environments. The new sensor is based on the principle of quadrature and has two output versions.

- The first version has two output signals 90° out of phase. Each output provides one pulse per target tooth.
- The second version has a speed signal that is twice the output pulses per revolution and it also has a direction signal. For example, the 2000 Series versions provide 60 symmetrical pulses per revolution with the 30-tooth target.

Outputs — Digital signals from NPN transistors (open collector output with internal 10K pull-up resistors).

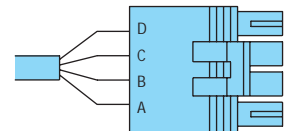
Supply Voltage: 8 to 24 Vdc* (compatible with 12V vehicle conditions)

Supply Current: 40 mA max. (Including internal pull-up resistors)

Output

Low Voltage: 0.5 Vdc maximum @ 10 mA

The sensor has reverse polarity protection, short circuit protection, load dump protection and EMC (Electromagnetic Compatibility) protection (the customer should qualify the EMC protection in their specific application).



Weatherpack Tower Connector

Connections —

Standard 4 prong Weatherpack connector with 18 AWG (American Wire Gage) cables or M12 threaded connector:

Weatherpack (Version 1)

- Position A (red) = power supply
- Position B (black) = common
- Position C (orange) = output one
- Position D (yellow) = output two

M12 Connector (Version 1)

- Pin 1 = power supply
- Pin 2 = output one
- Pin 3 = common
- Pin 4 = output two

Weatherpack (Version 2)

- Position A (red) = power supply
- Position B (black) = common
- Position C (blue) = speed signal
- Position D (white) = direction

M12 Connector (Version 2)

- Pin 1 = power supply
- Pin 2 = direction
- Pin 3 = common
- Pin 4 = speed signal

Note: The speed sensor or quadrature speed sensor option does **NOT** include read-out display. Possible sources for read-out display include:

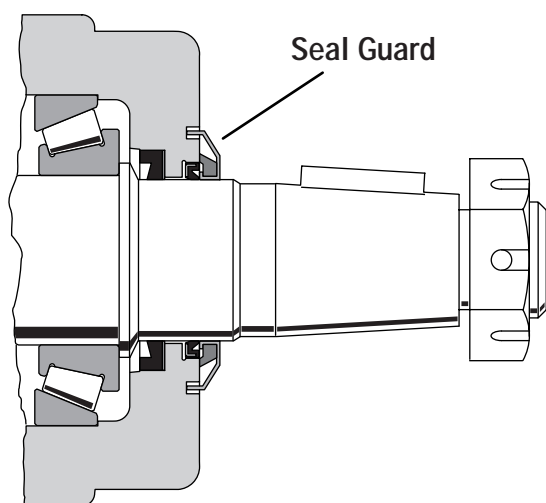
Eaton Corporation
Durant Products
 901 South 12th Street
 Watertown, WI 57094
 1-800-289-3866

Shaft Seal

This time proven shaft seal design has a patented feature which allows the seal lip to follow shaft deflection under high side loads and therefore provides better sealing. Additionally, these seals can withstand case pressure up to: 140 Bar [2000 PSI] 2000 Series, 100 Bar [1500 PSI] 4000 Series, 70 Bar [1000 PSI] 6000 Series, and 20 Bar [300 PSI] 10,000 Series.

To optimize seal life, reduce case pressures (with case drain) at shaft speeds greater than 250 RPM.

Optional Seal Guard Package for 2000, 4000, and 6000 Series



In response to the need for robust seal protection requirements, Eaton now offers a seal guard package. This feature consists of a metal shield that protects an internal wiper seal. The shield is interference-fit on the output shaft and rotates with the output shaft. For added protection, the shield is recessed into a special groove in the bearing housing face.

Centrifugal force causes foreign debris to be forced away from the high pressure shaft and dust seal area. The seal guard does not seal hydraulic fluid. Instead, it protects the standard seals from damage caused by foreign debris. Typical applications benefiting from this feature include street sweepers, industrial sweepers, and harvesting machinery.

NOTE: This option is used in conjunction with the special front retainer with shield groove. Special feature (Hardware) option code "28" for 2000, "13" for 4000, and "14" for 6000 Series, **these motors include the seal guard package, special front retainer and a special shaft with additional length** (6000 Series with design code -006 (effective December 1, 1995) will not require a special front retainer and standard shafts will accept the seal guard).

Internal Check Valves

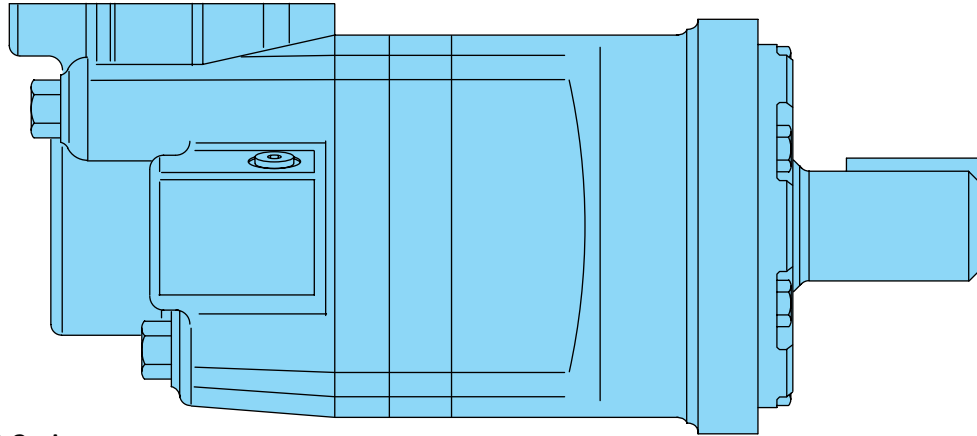
An internal check valve is provided to relieve case pressure to the low pressure side of the motor. This check valve system is adequate for most applications. In addition, motors have an external case pressure drain port for use when continuous back pressure exceeds: 140 Bar [2000 PSI] 2000 Series, 100 Bar [1500 PSI] 4000 Series, 70 Bar [1000 PSI] 6000 Series, and 20 Bar [300 PSI] 10,000 Series.

Corrosion Protected Disc Valve Motors

2000, 4000, 6000, and 10,000 Series motors are available with a corrosion resistant coating for use in hostile environments. This coating protects the motor from salt water, and various chemicals and is especially effective in marine, food processing, cleansing, fishing, and agricultural applications. Motor output shaft plating helps eliminate seal damage caused by these caustic or acid materials. Char-Lynn disc valve motors are available with just the output shaft plated, or with plated shaft and entire motor exterior coating.

Specifications

6000 Series



Specification Data—6000 Series

Displ. cm ³ /r [in ³ /r]		195 [11.9]	245 [15.0]	310 [19.0]	390 [23.9]	490 [30.0]	625 [38.0]	985 [60.0]	
Max. Speed (RPM) @ Flow	Continuous	775	615	485	387	307	241	153	
	Intermittent	866	834	698	570	454	355	230	
Flow LPM [GPM]	Continuous	150 [40]	150 [40]	150 [40]	150 [40]	150 [40]	150 [40]	150 [40]	
	Intermittent	170 [45]	210 [55]	225 [60]	225 [60]	225 [60]	225 [60]	225 [60]	
Torque Nm [lb-in]	★ 1-3/4 Inch Dia. Tapered Shaft	Continuous	575 [5100]	735 [6510]	930 [8230]	1155 [10230]	1445 [12800]	1480 [13100]	1685 [14920]
		Intermittent	860 [7620]	1100 [9740]	1355 [11990]	1635 [14490]	1885 [16670]	1898 [16800]	1875 [16580]
Pressure Δ Bar [Δ PSI]	★ 1-3/4 Inch Dia. Tapered Shaft	Continuous	205 [3000]	205 [3000]	205 [3000]	205 [3000]	205 [3000]	170 [2500]	140 [2000]
		Intermittent	310 [4500]	310 [4500]	310 [4500]	310 [4500]	275 [4000]	221 [3200]	140 [2000]
		Peak	310 [4500]	310 [4500]	310 [4500]	310 [4500]	310 [4500]	240 [3500]	170 [2250]

Maximum Case Pressure - without Case Drain * — 70 Bar [1000 PSI]

A simultaneous maximum torque and maximum speed NOT recommended. For permissible continuous and intermittent operating combinations of pressure and flow refer to performance data on pages 52-53.

★ **Maximum torque for 1-1/2 inch shaft** — 1325 Nm [11750 lb-in] Continuous and 1650 Nm [14600 lb-in] intermittent.

* For back pressure over 70 Bar [1000 PSI] use an external case drain. Install case drain lines so that the motor case remains filled at all times.

Maximum inlet pressure — 310 Bar [4500 PSI]. Do not exceed Δ pressure rating (see chart above).

* **Maximum return pressure** — 310 Bar [4500 PSI]. Do not exceed Δ pressure rating (see chart above).

Δ Bar [Δ PSI] — True pressure difference between inlet port and outlet port.

Continuous Rating — Motor may be run continuously at these ratings.

Intermittent Operation — 10% of every minute.

Peak Operation — 1% of every minute.

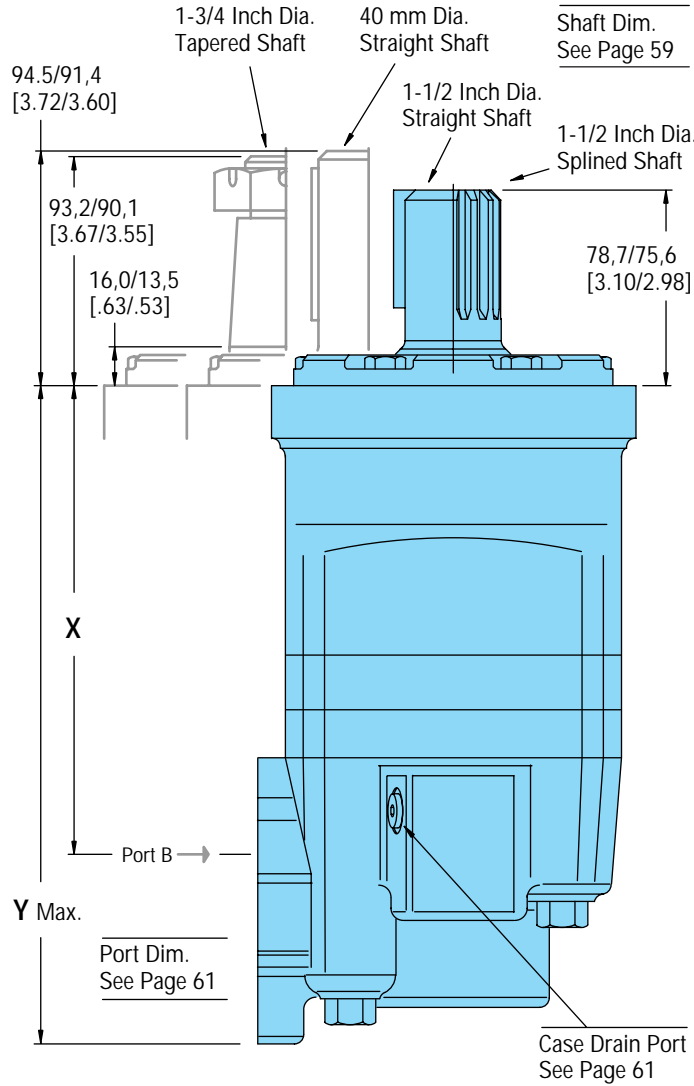
Recommended Fluids — Premium quality, anti-wear type hydraulic oil with a viscosity of not less than 70 SUS at operating temperature (see page 81).

Recommended Maximum System Operating Temp. — Is 82° C [180° F]

Recommended Filtration — per ISO Cleanliness Code, level 18/13

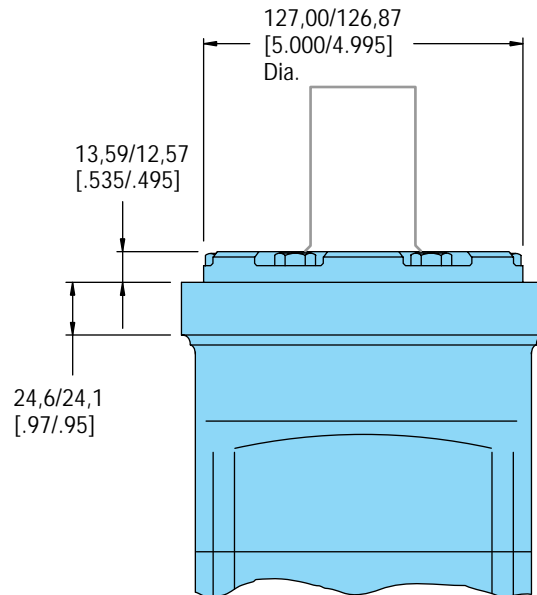
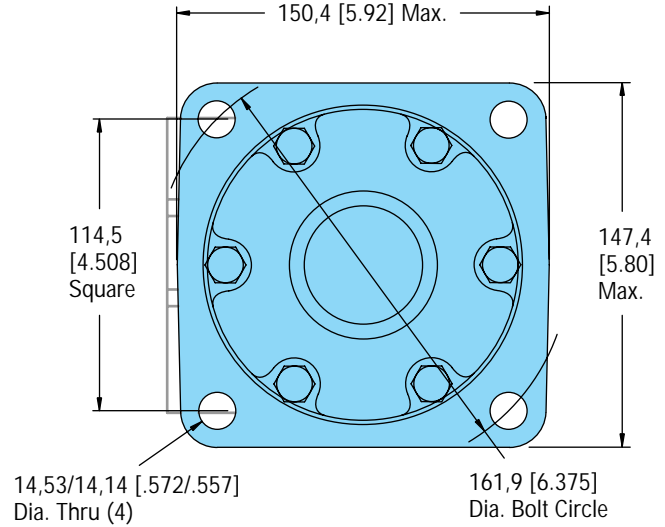
To assure best motor life, run motor for approximately one hour at 30% of rated pressure before application to full load. Be sure motor is filled with fluid prior to any load applications.

Dimensions — 6000 Series Standard Motor



Standard Rotation
Viewed from Shaft End
Port A Pressurized — CW
Port B Pressurized — CCW

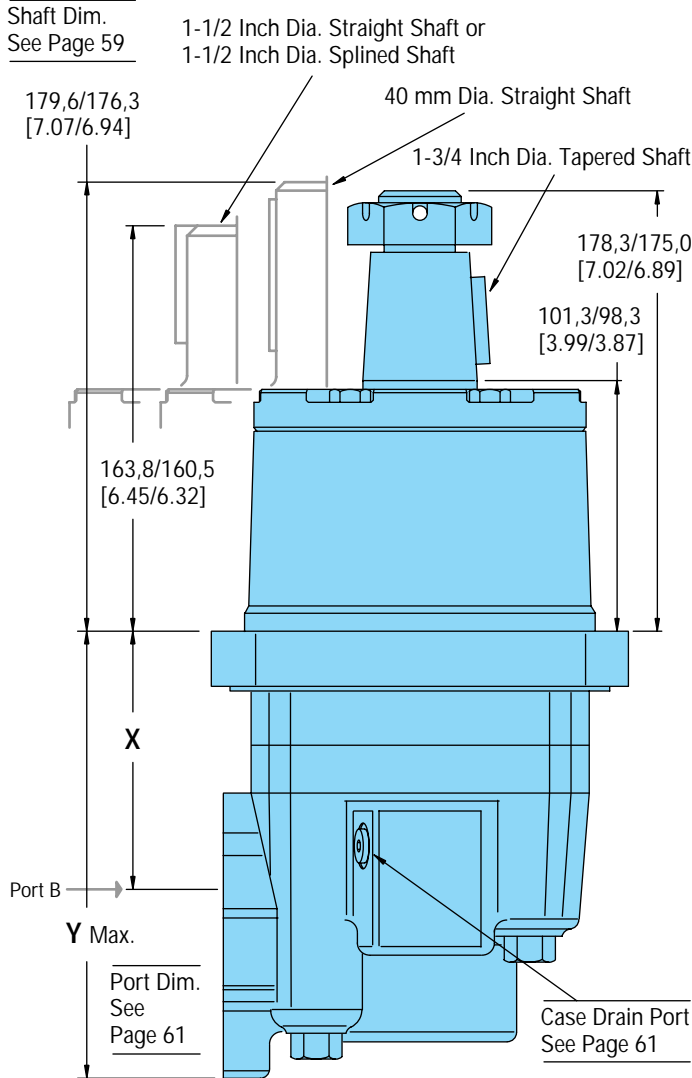
Standard SAE CC Flange



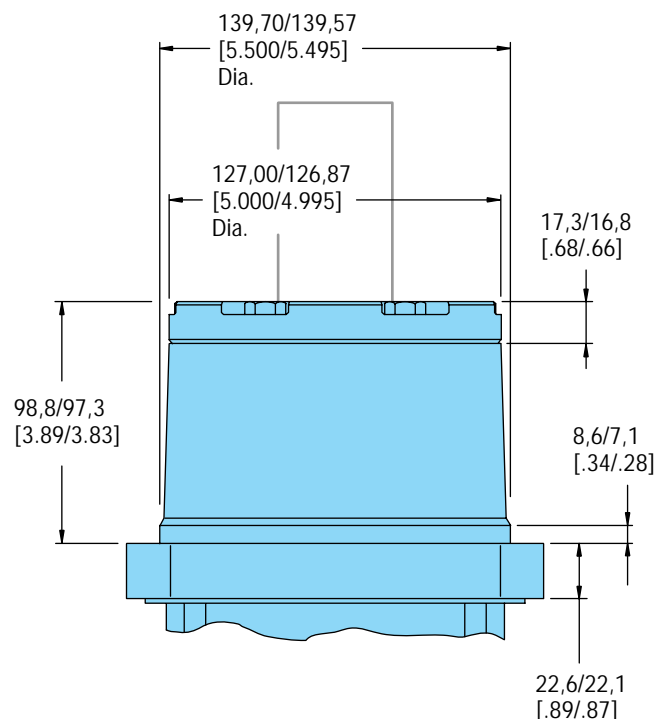
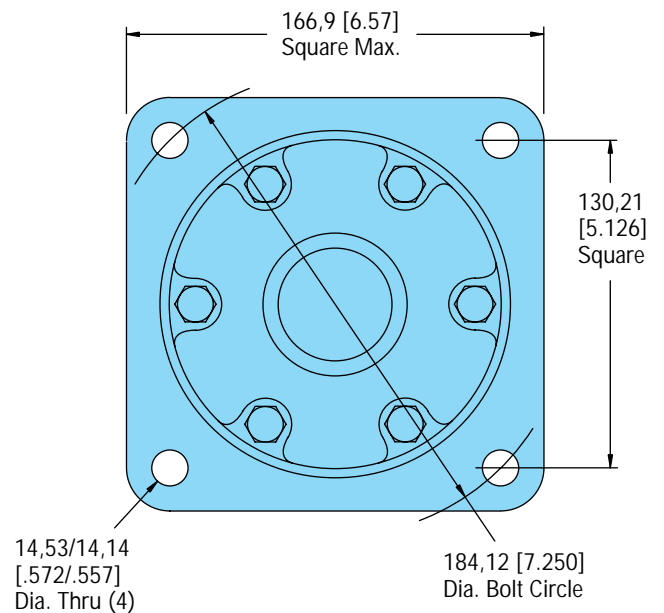
6000 Series Standard Motor with 1-5/16-12 O-ring Ports, G 1 (BSP) Ports, and 3/4 inch Split Flange Ports

Displ. cm ³ /r [in ³ /r]	195 [11.9]	245 [15.0]	310 [19.0]	390 [23.9]	490 [30.0]	625 [38.0]	985 [60.0]
X Dim. mm [inch]	187,7 [7.39]	193,3 [7.61]	200,7 [7.90]	209,3 [8.24]	220,5 [8.68]	235,2 [9.26]	274,8 [10.82]
Y Dim. mm [inch]	270,1 [10.63]	275,6 [10.85]	283,0 [11.14]	291,6 [11.48]	302,8 [11.92]	318,5 [12.54]	357,4 [14.07]

Dimensions — 6000 Series Wheel Motor



Standard Rotation
Viewed from Shaft End
Port A Pressurized — CW
Port B Pressurized — CCW



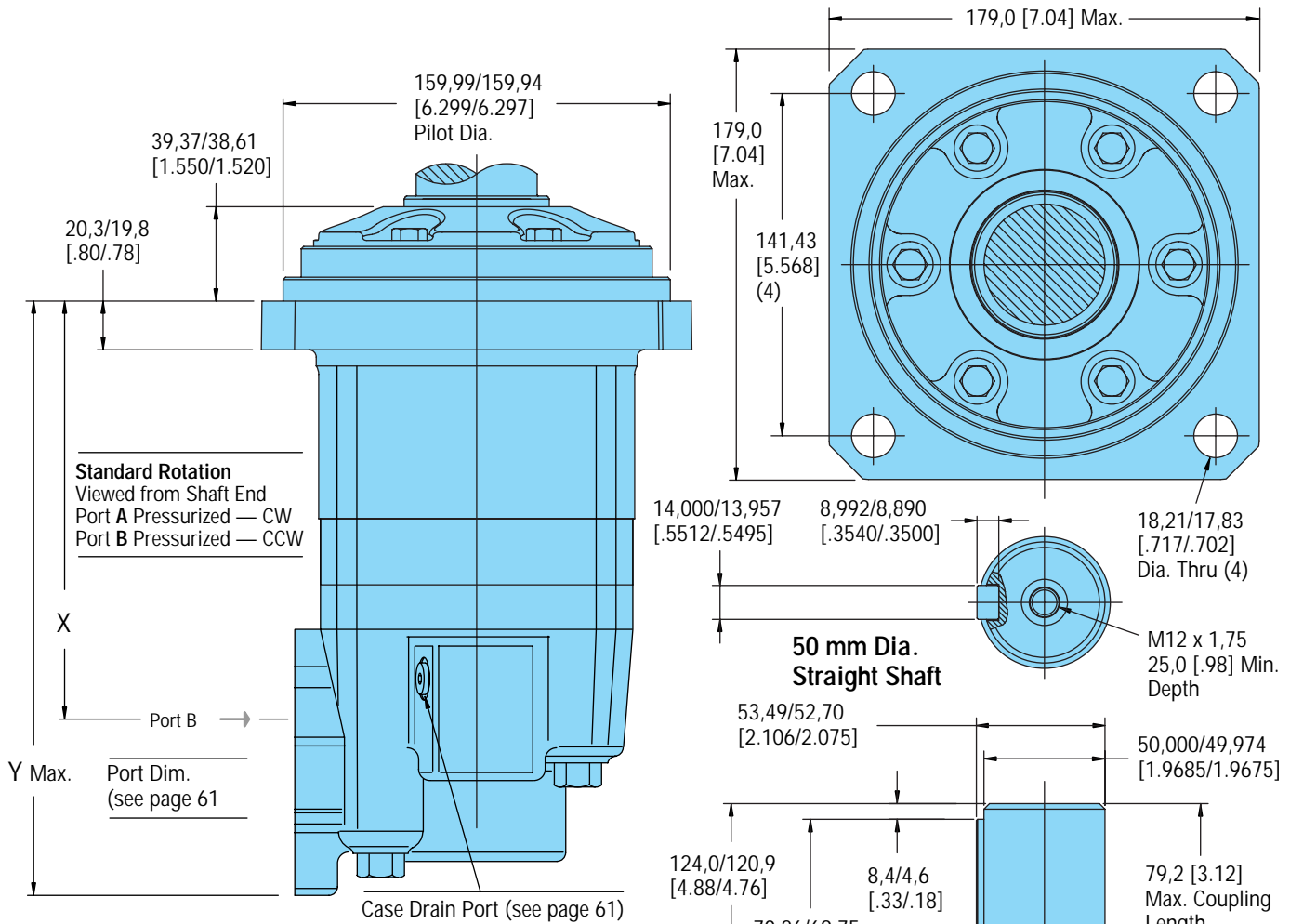
6000 Series Wheel Motor with 1-5/16-12 O-ring Ports, G 1 (BSP) Ports, and 3/4 inch Split Flange Ports

Displ. cm ³ /r [in ³ /r]	195 [11.9]	245 [15.0]	310 [19.0]	390 [23.9]	490 [30.0]	625 [38.0]	985 [60.0]
X Dim. mm [inch]	102,9 [4.05]	108,4 [4.27]	115,8 [4.56]	124,7 [4.91]	135,6 [5.34]	149,9 [5.90]	190,0 [7.48]
Y Dim. mm [inch]	185,2 [7.29]	190,8 [7.51]	198,2 [7.80]	207,1 [8.15]	218,0 [8.58]	233,0 [9.17]	272,6 [10.73]

Dimensions — 6000 Series

Global Mount

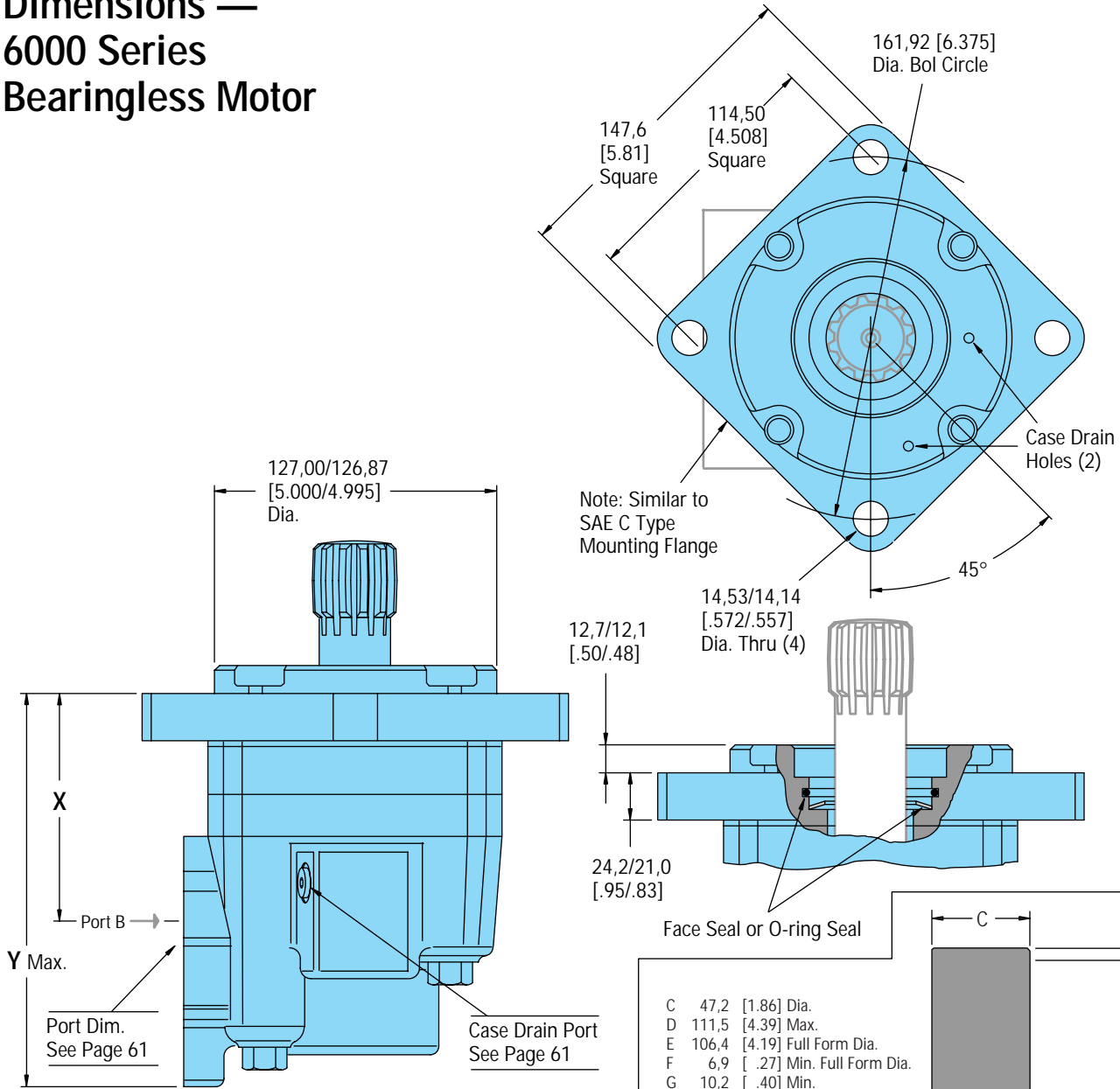
(Similar to ISO 3019/2)



6000 Series Standard Motor with G 1 (BSP) Ports

Displ. cm ³ /r [in ³ /r]	310 [19.0]	390 [24.0]	490 [30.0]	625 [38.0]	800 [45.0]	800 [49.0]	985 [60.0]
X Dim. mm [inch]	182,4 [7.18]	191,0 [7.52]	202,2 [7.96]	216,7 [8.54]	229,4 [9.03]	236,7 [9.32]	256,5 [10.10]
Y Dim. mm [inch]	262,6 [10.34]	271,5 [10.69]	282,4 [11.12]	297,2 [11.70]	309,6 [12.19]	317,0 [12.48]	337,0 [13.27]

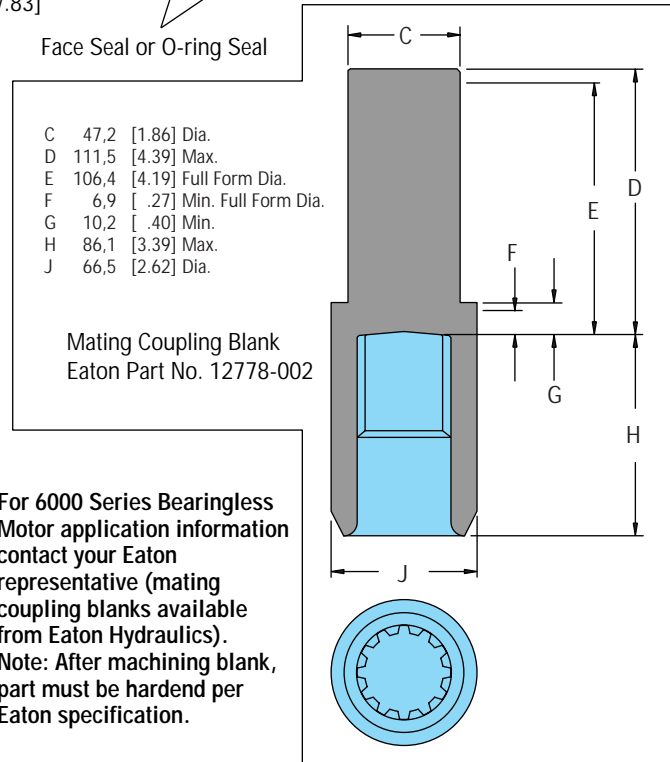
Dimensions — 6000 Series Bearingless Motor



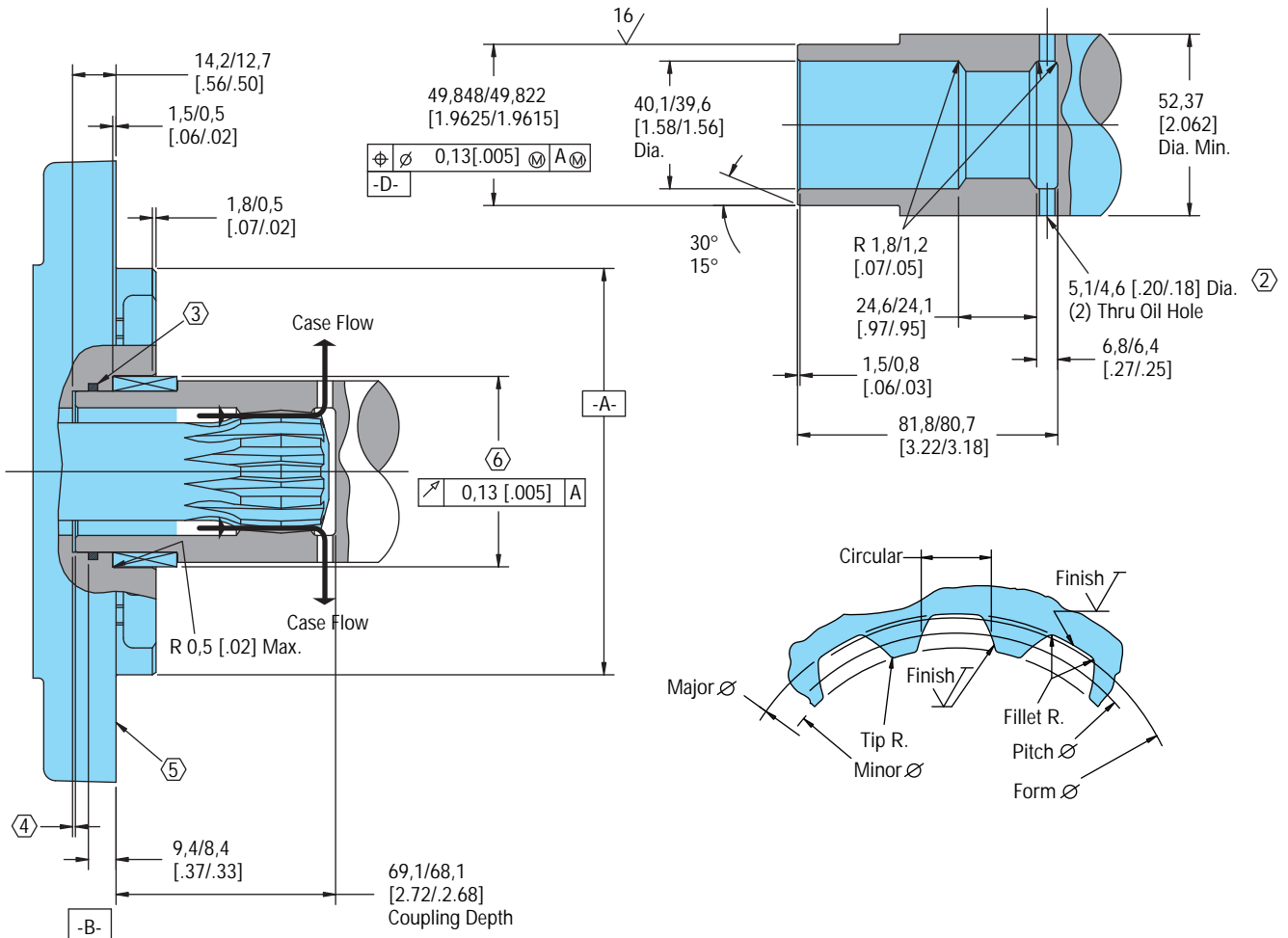
Standard Rotation
Viewed from Drive End
Port A Pressurized — CW
Port B Pressurized — CCW

6000 Series Bearingless Motor with 1-5/16-12 O-ring Ports,
G 1 (BSP) Ports, 3/4 inch Split Flange Ports

Displ. cm ³ /r [in ³ /r]	195 [11.9]	245 [15.0]	310 [19.0]	390 [23.9]	490 [30.0]	625 [38.0]	985 [60.0]
X Dim. mm [inch]	105,7 [4.16]	111,2 [4.38]	118,3 [4.66]	127,5 [5.02]	138,7 [5.46]	152,9 [6.02]	193,0 [7.60]
Y Dim. mm [inch]	188,0 [7.40]	193,6 [7.62]	200,7 [7.90]	209,6 [8.25]	220,8 [8.69]	235,5 [9.27]	275,1 [10.83]



Bearingless Installation — 6000 Series



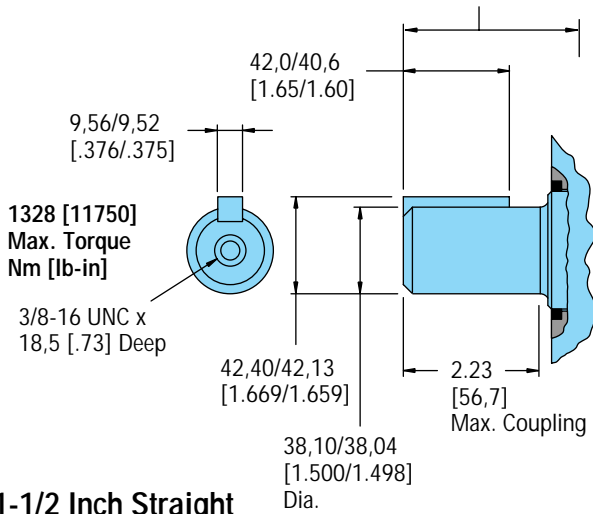
Spline Pitch — 8.5/17
 Pressure Angle — 30°
 Number of teeth — 12
 Class of Fit — Ref. 5
 Type of Fit — Side
 Pitch Diameter — Ref. 35,858823 [1.4117647] $\text{Ⓢ} 0,20 [0.008] H$
 Base Diameter — Ref. 31,054652 [1.2226241]
 Major Diameter — 39,17 [1.542] Max. 38,97 [1.534] Min.
 Minor Diameter — 33,30 - 33,48 [1.311 - 1.318]
 Form Diameter, Min. — 38,33 [1.509]
 Fillet Radius — 0,64 - 0,76 [0.025 - .030]
 Tip Radius — 0,25 - 0,51 [0.010 - .020]
 Finish — 1,6 [63]
 Involute Profile Variation — +0,000 -0,025 [+0.0000 - .0010]
 Total Index Variation — 0,038 [0.0015]
 Lead Variation — 0,013 [0.0005]
 Circular Space Width:
 Maximum Actual — 5,898 [0.2322]
 Minimum Effective — 5,804 [0.2285]
 Maximum Effective — Ref. 5,857 [0.2306]
 Minimum Actual — Ref. 5,834 [0.2297]
 Dimension Between Two Pins — Ref. 26,929 - 27,084 [1.0602 - 1.0663]
 Pin Diameter — 6,223 [0.2450] Pins to Have 4,0 [0.160] Wide Flat for Root Clearance

- 1 Internal spline in mating part to be as follows:
 Material to be ASTM A304, 8620H.
 Carborize to a hardness of 60-64 HRC with case depth (to 50HRC) of 0,076 - 1,02 [0.030 - .040] (dimensions apply after heat treat).
- 2 Mating part to have critical dimensions as shown. Oil holes must be provided and open for proper oil circulation.
- 3 Some means of maintaining clearance between shaft and mounting flange must be provided.
- 4 Seal to be furnished with motor for proper oil circulation thru splines.
- 5 Similar to SAE "C" Four Bolt Flange.
- 6 Counterbore designed to adapt to a standard sleeve bearing 50,010 - 50,038 [1.9689 - 1.9700] ID by 60,051 - 60,079 [2.3642 - 2.3653] O.D. (Oilite bronze sleeve bearing).

Dimensions — Shafts

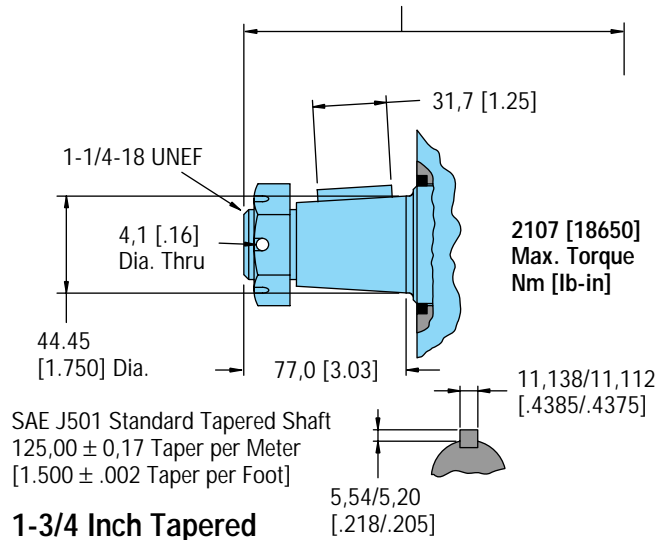
6000 Series

78,6/75,8 [3.10/2.98] End of Shaft to Mounting Surface (Std)



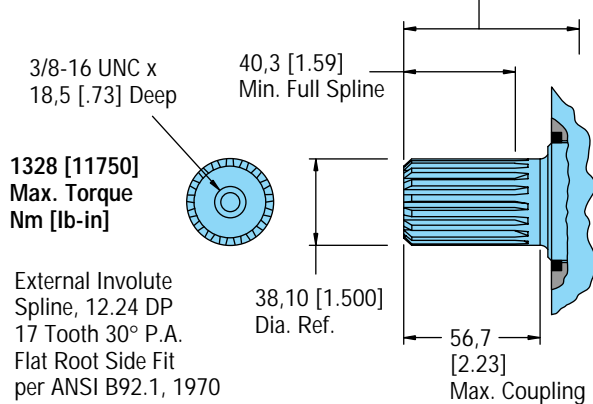
1-1/2 Inch Straight

178,1/175,0 [7.02/6.89] End of Shaft to Mounting Surface (Whl)

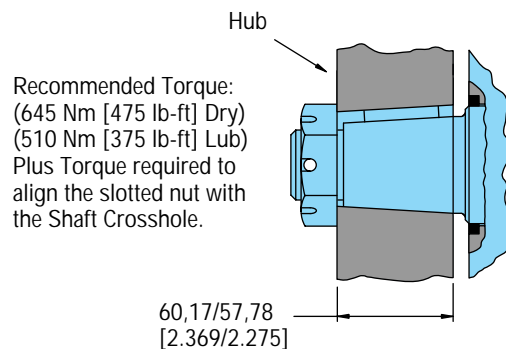


1-3/4 Inch Tapered

78,6/75,8 [3.10/2.98] End of Shaft to Mounting Surface (Std)

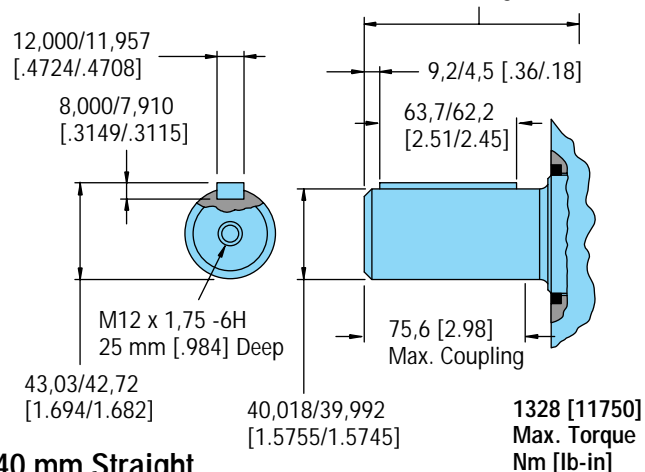


1-1/2 Inch 17 Tooth Splined

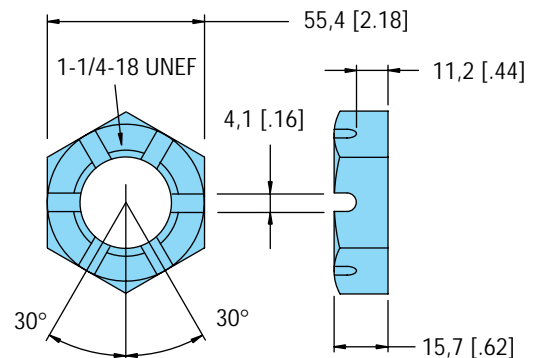


Tapered Shaft Hub Data

94,4/91,6 [3.72/3.60] End of Shaft to Mounting Surface (Std)



40 mm Straight



Slotted Hexagon Nut

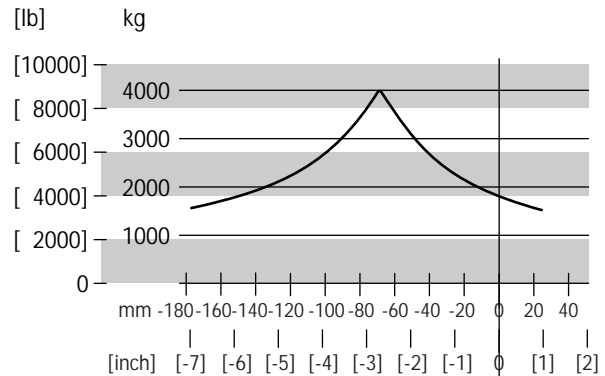
Shaft Side Load Capacity 6000 Series

These curves indicate the radial load capacity on the motor shaft(s) at various locations.

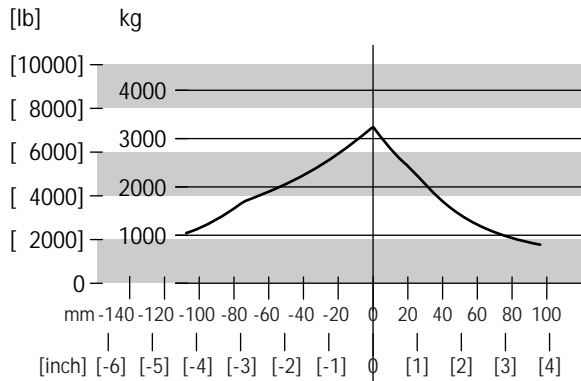
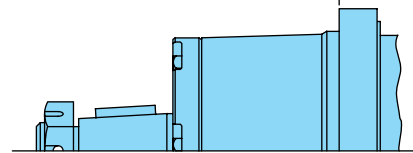
The curve is based on B 10 Bearing life (2000 hours or 12,000,000 shaft revolutions at 100 RPM) at rated output torque. To determine radial load at speeds other than 100 RPM, multiply the load values given on the bearing curve by the factors in the chart below.

RPM	Multiplication Factor
50	1.23
100	1.00
200	.81
300	.72
400	.66
500	.62
600	.58
700	.56
800	.54

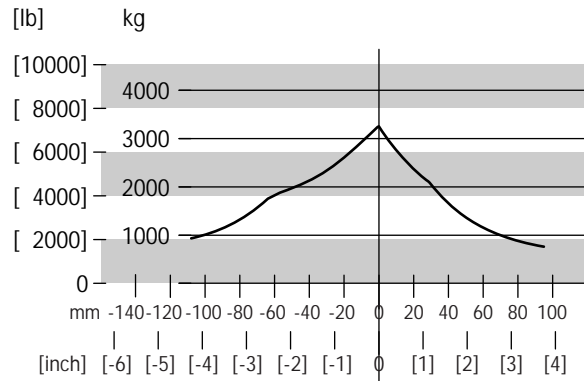
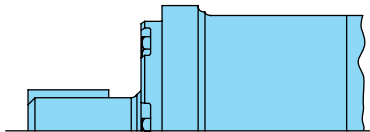
For 3,000,000 Shaft revolutions or 500 hours — Increase these shaft loads 52%.



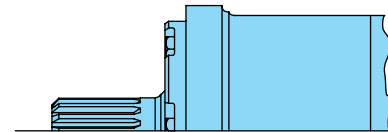
Wheel Motor
Tapered
Shaft



Standard Motor
Straight
Shaft



Standard Motor
Splined
Shaft



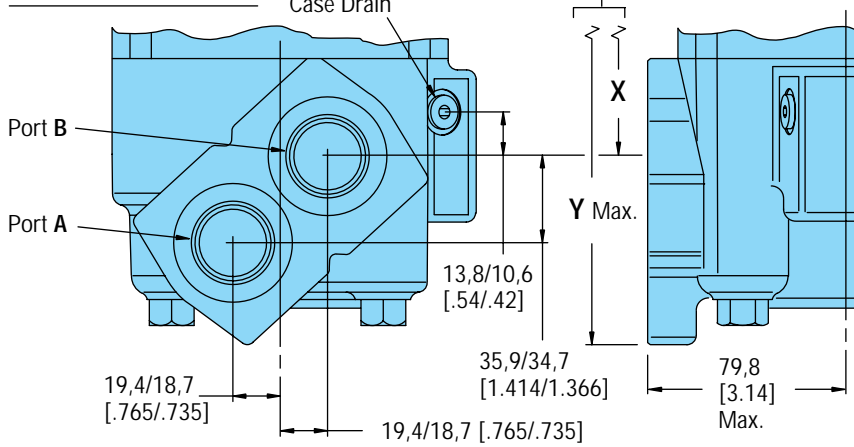
Dimensions — Ports 6000 Series

6000 Series with Shuttle Valve

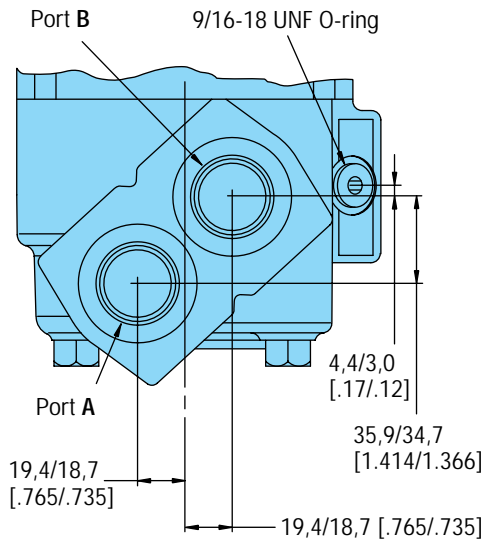
1-5/16-12 O-ring Ports (2)
or G 1 (BSP) Ports (2)

7/16-20 UNF O-ring or
G 1/4 (BSP) Port
Case Drain

See Pages 54-57



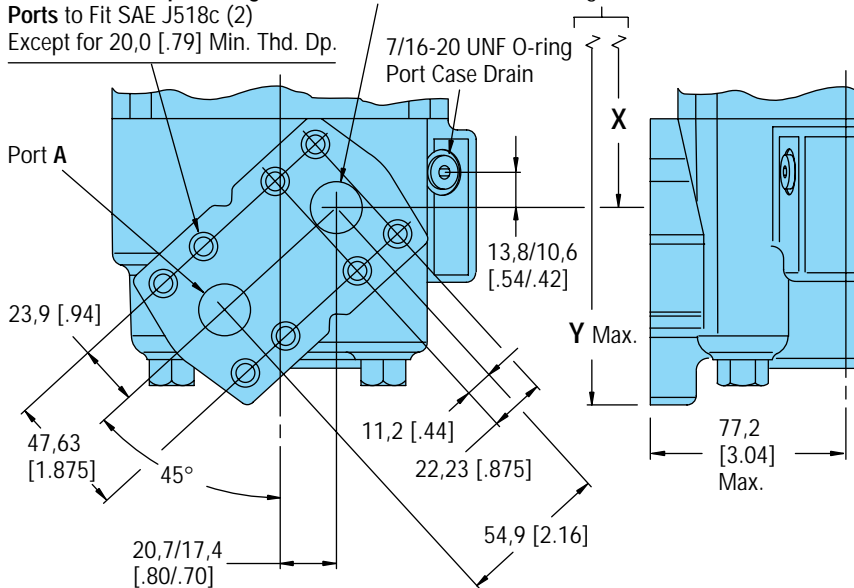
1-5/16-12 O-ring Ports (2) with Shuttle



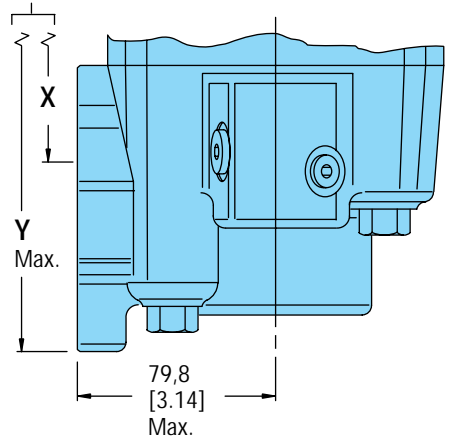
4 Bolt 3/4 Inch Split Flange
Ports to Fit SAE J518c (2)
Except for 20,0 [.79] Min. Thd. Dp.

Port B

See Pages 54-57



See Pages 54-57

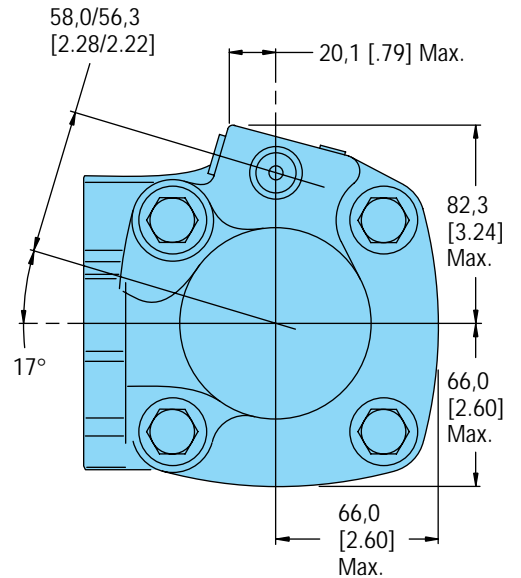
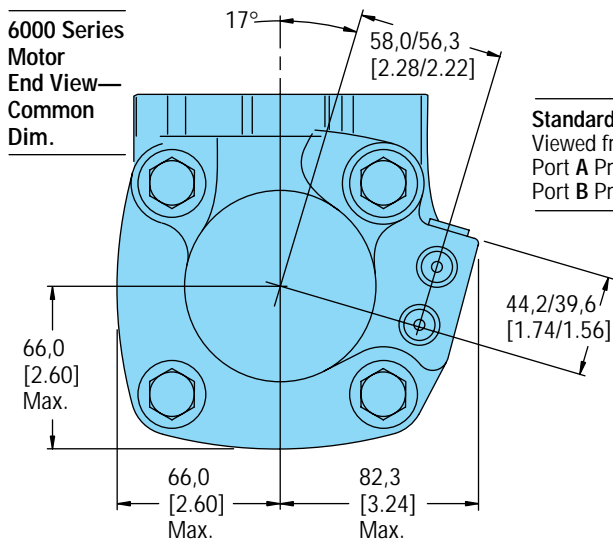


6000 Series
Motor
End View—
Common
Dim.

17°

58,0/56,3
[2.28/2.22]

Standard Rotation — 6000 Series
Viewed from Shaft End
Port A Pressurized — CW
Port B Pressurized — CCW



Product Numbers 6000 Series

Product Numbers—6000 Series Motors

Use digit prefix —112-, 113-, or 114- plus four digit number from charts for complete product number—Example 114-1047.
Orders will not be accepted without three digit prefix.

Mounting	Shaft	Ports	Displacement cm ³ /r [in ³ /r] and Product Number						
			195 [11.9]	245 [15.0]	310 [19.0]	390 [23.9]	490 [30.0]	625 [38.0]	985 [60.0]
Standard	1-1/2 Inch Straight	1-5/16 O-ring	112-1064	-1065	-1066	-1067	-1068	-1107	-1069
	40 mm Straight	G 1 (BSP)	112-1094	-1095	-1096	-1097	-1098	—	-1099
	1-1/2 Inch 17 T Splined	1-5/16 O-ring	112-1058	-1059	-1060	-1061	-1062	-1109	-1063
		G 1 (BSP)	112-1088	-1089	-1090	-1091	-1092	—	-1093
Wheel Motor	40 mm Straight	G 1 (BSP)	113-1082	-1083	-1084	-1085	-1086	-1100	-1087
	1-3/4 Inch Tapered	1-5/16 O-ring	113-1070	-1071	-1072	-1073	-1074	-1093	-1075
Bearingless		1-5/16 O-ring	114-1031	-1032	-1033	-1034	-1035	-1055	-1036
		G 1 (BSP)	114-1043	-1044	-1045	-1046	-1047	—	-1048

114-1047

Product Numbers – 6000 Series

Mounting Type - Standard (Code H), 4 Bolt:

- 160,0 [6.30] Pilot Dia.
- 18,01 [.709] Dia. Mounting Holes
- 200,0 [7.87] Dia. Bolt Circle

Output Shaft - Straight (Code 21)

Ports - G1 (BSP) Staggered G 1/4 Case Drain (Code C)

Paint - Low Gloss Black (Code A)

Use digit prefix —112- plus four digit number from charts for complete product number—Example 112-1215.
Orders will not be accepted without three digit prefix.

112-1215

Mounting	Shaft	Ports	Displacement cm ³ /r [in ³ /r] and Product Number						
			310 [19.0]	390 [23.9]	490 [30.0]	625 [38.0]	737 [45.0]	800 [49.0]	985 [60.0]
Standard	50 mm Straight	G 1 (BSP)	112-1217	-1218	-1215	-1216	-1247	-1219	-1220

For 6000 Series Motors with a configuration *Not Shown* in the charts above: Use model code number system on page 63 to specify product in detail.

Model Code for 6000 Series Motors

The following 14-digit coding system has been developed to identify all of the configuration options for the 6000 Series Motor. Use this model code to specify a motor with the desired features. All 14-digits of the code must be present when ordering. You may want to photocopy the matrix below to ensure that each number is entered in the correct box.

Model Code — 6000 Series Disc Valve Motors

1	2	3	4	5	6	7	8	9	10	11	12	13	14
M	0	6										0	0

Position 1 Product Series

M..... Motor

Position 2, 3 6000 Series

06..... 6000 Series

Position 4, 5 Displacement cm³/r [in³/r]

12..... 195 [11.9]

15..... 245 [15.0]

19..... 310 [19.0]

24..... 390 [23.9]

30..... 490 [30.0]

38..... 625 [38.0]

45..... 740 [45.0]*

49..... 805 [49.0]*

60..... 985 [60.0]

* For performance and dimension data contact your Eaton Hydraulics representative.

Position 6 Mounting Flange

A..... 4 Bolt (Bearingless 127,0 [5.00] Pilot Dia. and 14,3 [.56] Dia. Mounting Holes 161,9 [6.38] Dia. B.C.

B..... 4 Bolt (SAE CC) (Standard) 127,0 [5.00] Pilot Dia. and 14,3 [.56] Mounting Holes on 161,9 [6.38] Dia. B.C.

C..... 4 Bolt (Wheel) 139,7 [5.50] Pilot Dia. and 14,3 [.56] Dia. Mounting Holes on 184,1 [7.25] Dia. B.C.

H..... 4 Bolt (Global) (Standard) 160,0 [6.30] Pilot Dia. and 18,0 [.709] Dia. Mounting Holes on 200,0 [7.87] Dia. Bolt Circle

Position 7, 8 Output Shaft

00..... Bearingless

01..... 1-1/2 inch Dia. Straight with Straight Key, 3/8-16 Threaded Hole and 56,7 [2.23] Max. Coupling Length

02..... 1-3/4 inch Dia. Tapered with Straight Key and 1-1/4 - 18 UNEF Slotted Hex. Nut

03..... 1-1/2 inch Dia. Splined 17T with 40,3 [1.59] Min. Full Spline Length and 3/8-16 Threaded Hole

10..... 40 mm Dia. Straight with Straight Key, M12 x 1,75-6H Threaded Hole

21..... 50 mm Dia. Straight with Straight Key, 14W x 9H x 70L, M12 x 1,75-6H Thread in End (Available with Mounting Flange Code H Only)

24..... 60 mm Dia. 10:1 Tapered Shaft with M42 x 3-6H Threaded End (Available with Mounting Flange Code H Only)

25..... 2 1/8 inch Dia. Splined 16 Tooth, 55,9 mm [2.20] Min. Full Spline Length (Available with Mounting Flange Code H Only)

Position 9 Port Type

A..... 1-5/16 - 12 O-ring with 7/16-20 O-ring Case Drain and Check Valve

B..... 3/4 inch 4 Bolt Split Flange with 7/16-20 O-ring Case Drain and Check Valve

C..... G 1 (BSP) O-ring with G 1/4 (BSP) O-ring Case Drain and Check Valve

D..... 1- 5/16-12 O-ring with (2) 9/16-18 O-ring Case Drain Ports and Hot Oil Shuttle Valve

R..... 1- 5/16-12 O-ring with (2) 9/16-18 O-ring Ports for External Lubrication Circuit (both case ports must be connected)

Position 10, 11 Special Features (Hardware)

00..... None

10..... Viton Shaft Seal

07..... Viton Seals

13..... Reverse Rotation

14..... Seal Guard

21..... Speed Sensor (Std.)

Position 12 Paint/Special Packaging

0..... No Paint

A..... Painted Low Gloss Black

B..... Corrosion Protected

Position 13 Eaton Assigned Code when Applicable

0..... Assigned Code

Position 14 Eaton Assigned Design Code

0..... Assigned Design Code

Fluid Recommendations

Char-Lynn Disc Valve Motors

Introduction

The ability of Eaton hydraulic components to provide the desired performance and life expectancy depends largely on the fluid used. The purpose of this section is to provide readers with the knowledge required to select the appropriate fluids for use in systems that employ Eaton hydraulic components.

One of the most important characteristics to consider when choosing a fluid to be used in a hydraulic system is viscosity. Viscosity choice is always a compromise; the fluid must be thin enough to flow easily but thick enough to seal and maintain a lubricating film between bearing and sealing surfaces. See chart below for viscosity requirements.

Viscosity and Temperature

Fluid temperature affects viscosity. In general, as the fluid warms it gets thinner and its viscosity decreases. The opposite is true when fluid cools. When choosing a fluid, it is important to consider the start-up and operating temperatures of the hydraulic system. Generally, the fluid is thick when the hydraulic system is started. With movement, the fluid warms to a point where a cooling system begins to operate. From then on, the fluid is maintained at the temperature for which the hydraulic system was designed. In actual applications this sequence varies; hydraulic systems are used in many environments from very cold to very hot. Cooling systems also vary from very elaborate to very simple, so ambient temperature may affect operating temperature. Equipment manufacturers who use Eaton hydraulic components in their products should anticipate temperature in their designs and make the appropriate fluid recommendations to their customers.

Cleanliness

Cleanliness of the fluid in a hydraulic system is extremely important. Eaton recommends that the fluid used in its hydraulic components be maintained at ISO Cleanliness Code 18/13 per SAE J1165. This code allows a maximum of 2500 particles per milliliter greater than 5 μm and a maximum of 80 particles per milliliter greater than 15 μm . Cleanliness requirements for specific products are given in the table below. OEM's and distributors who use Eaton hydraulic components in their products should provide for these requirements in their designs. A reputable filter supplier can supply filter information.

Char-Lynn Disc Valve Motors	Viscosity		ISO Cleanliness Requirements
	Minimum	Best Range	
	70 SUS 13 cSt	100-200 SUS 20-43 cSt	18/13

Additional Notes:

- Fluids too thick to flow in cold weather start-ups will cause pump cavitation and possible damage. **Motor cavitation is not a problem during cold start-ups (with one exception — two speed motors).**
- Minimum / Maximum operating temperatures are -29° C / 82° C [-20° F / 180° F].
- When choosing a hydraulic fluid, all the components in the system must be considered and the best viscosity range adjusted accordingly. For example, when a medium duty piston pump is combined with a Geroler motor the best viscosity range becomes

Fluid Maintenance

Maintaining correct fluid viscosity and cleanliness level is essential for all hydraulic systems. Since Eaton hydraulic components are used in a wide variety of applications it is impossible for Eaton to publish a fluid maintenance schedule that would cover every situation. Field testing and monitoring are the only ways to get accurate measurements of system cleanliness. OEM's and distributors who use Eaton hydraulic components should test and establish fluid maintenance schedules for their products. These maintenance schedules should be designed to meet the viscosity and cleanliness requirements laid out in this document.

Fluid Selection

Premium grade petroleum based hydraulic fluids will provide the best performance in Eaton hydraulic components. These fluids typically contain additives that are beneficial to hydraulic systems. **Eaton recommends fluids that contain anti-wear agents, rust inhibitors, anti-foaming agents, and oxidation inhibitors.** Premium grade petroleum based hydraulic fluids carry an ISO VG rating.

SAE grade crankcase oils may be used in systems that employ Eaton hydraulic components, but it should be noted that these oils may not contain all of the recommended additives. This means using crankcase oils may increase fluid maintenance requirements.

Hydraulic fluids that contain V.I. (viscosity index) improvers, sometimes called multi-viscosity oils, may be used in systems that employ Eaton hydraulic components. These V.I. improved fluids are known to "shear-down" with use. This means that their actual viscosity drops below the rated value. Fluid maintenance must be increased if V.I. improved fluids are used. Automotive automatic transmission fluids contain V.I. improvers.

Synthetic fluids may be used in Eaton hydraulic components. A reputable fluid supplier can provide information on synthetic fluids. Review applications that require the use of synthetic fluids with your Eaton representative.

100 - 150 SUS [20 - 32 cSt] and viscosity should never fall below 70 SUS [13 cSt].

- If the natural color of the fluid has become black it is possible that an overheating problem exists.
- If the fluid becomes milky a water contamination problem may exist.
- Take fluid level reading when the system is cold.
- Contact your Eaton representative if you have specific questions about the fluid requirements of Eaton hydraulic components.

Motor Application Information — Vehicle Drive Calculations

Step One — Calculate Motor Speed (RPM)

$$\text{RPM} = \frac{2.65 \times \text{KPH} \times G}{R_m} \quad \text{RPM} = \frac{168 \times \text{MPH} \times G}{R_1}$$

where KPH = vehicle speed (kilometers per hour)
 where MPH = vehicle speed (miles per hour)
 R_m = rolling radius of tires (meter)
 R₁ = rolling radius of tires (inch)
 G = gear reduction ratio (if any) between motors and wheels. If no gear box or other gear reduction devices are used G = 1.

If vehicle speed is expressed in m/second, multiply by 3.6 to convert to KPH.

If vehicle speed is expressed in ft./second, divide by 1.47 to convert to MPH.

Step Two — Determine Rolling Resistance

Rolling resistance (RR) is the force required to propel a vehicle over a particular surface. The values in Table 1 are typical of various surfaces per 1000 lb. of vehicle weight.

$$\text{RR} = \text{GVW} \times \rho \text{ (kg) (lb)}$$

where GVW = gross (loaded) vehicle weight lb/Kg
 ρ = value from Table 1

Table 1 - Rolling Resistance Coefficients for Rubber Tires on Various Surfaces

Surface	ρ
Concrete, excellent	.010
Concrete, good	.015
Concrete, poor	.020
Asphalt, good	.012
Asphalt, fair	.017
Asphalt, poor	.022
Macadam, good	.015
Macadam, fair	.022
Macadam, poor	.037
Snow, 2 inch	.025
Snow, 4 inch	.037
Dirt, smooth	.025
Dirt, sandy	.040
Mud	.037 to .150
Sand, Gravel	.060 to .150
Sand, loose	.160 to .300

Step Three — Tractive Effort to Ascend Grade

The largest grade a vehicle can ascend is called its "gradability." Grade is usually expressed as a percent rather than in degrees. A rise of one meter in ten meters or one foot rise in ten feet of travel is a 1/10 or 10 percent grade.

$$\text{GR} = \text{GVW} (\sin \theta + \rho \cos \theta)$$

Table 2

Comparison Grade (%)	Table Slope (Degrees)
1%	0°35'
2%	1° 9'
5%	2°51'
6%	3°26'
8%	4°35'
10%	5°43'
12%	6°5'
15%	8°31'
20%	11°19'
25%	14° 3'
32%	18°
60%	31°

Step Four — Determine Acceleration Force (FA)

The force (FA) required to accelerate from stop to maximum speed (KPH) or (MPH) in time (t) seconds can be obtained from the following equation:

Step Five — Determine Drawbar Pull

Drawbar Pull (DP) is total force available at the drawbar or "hitch" after the above forces have been subtracted from the total propelling force produced by the hydraulic motors. This value is established as either:

$$\text{FA} = \frac{\text{KPH} \times \text{GVW} \text{ (kg)}}{3.6 \text{ t}}$$

FA = Acceleration Force (Newton)
 t = Time (Seconds)

$$\text{FA} = \frac{\text{MPH} \times \text{GVW} \text{ (lb)}}{22 \text{ t}}$$

FA = Acceleration Force (lb)
 t = Time (Seconds)

1. A goal or objective of the designer.
2. A force required to pull a trailer (Repeat steps two through four above using trailer weight and add the three forces together to obtain DP).

Step Six — Total Tractive Effort

The tractive effort (TE) is the total force required to propel the vehicle and is the sum of the forces determined in Steps 2 through 5.

$$TE = RR + GR + FA + DP \text{ (Kg. or lb.)}$$

_____ Drawbar pull desired
 _____ Force required to accelerate
 _____ Force required to climb a grade
 _____ Force required to overcome rolling resistance

Wind resistance forces can usually be neglected. However, it may be wise to add 10% to the above total to allow for starting resistances caused by friction in bearings and other mechanical components.

Step Seven — Calculate Hydraulic Motor Torque (T)

$$T = \frac{TE \times R_m}{N \times G \times E_g} \text{ (Nm / Motors)}$$

$$T = \frac{TE \times R_l}{N \times G \times E_g} \text{ (lb-in / Motors)}$$

where N = number of driving motors
 Eg = gear box mechanical efficiency

Step Eight—Wheel Slip

If the torque required to slip the wheel (TS) is less than the torque calculated in Step 7, the performance objectives cannot be achieved.

$$TS = \frac{W \times f \times R_m}{G \times E_g} \text{ (Nm / Motor)}$$

$$TS = \frac{W \times f \times R_l}{G \times E_g} \text{ (lb-in / Motor)}$$

Where: f = coefficient of friction
 W = loaded vehicle weight over drive wheel

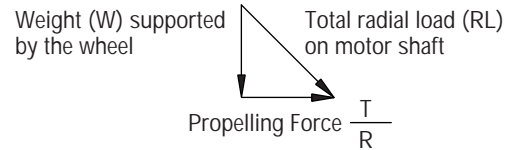
Coefficient of friction (f)

Steel on steel	0.15 to 0.20
Rubber tire on dirt	0.5 to 0.7
Rubber tire on asphalt	0.8 to 1.0
Rubber tire on concrete	0.8 to 1.0
Rubber tire on grass	0.4

It may be desirable to allow the wheel to slip to prevent hydraulic system overheating when excessive loads are imposed should the vehicle stall. In this case TS should be just slightly larger than T.

Step Nine — Motor Radial Load Carrying Capacity

When a motor is used to drive a vehicle with the wheel mounted directly on the motor shaft or rotating hub, the Total Radial Load (RL) acting on the motor shaft is the vector summation of two forces acting at right angles to each other.



$$RL = \sqrt{W^2 + \left(\frac{T}{R}\right)^2}$$

Refer to radial load rating of each motor (see table of catalog contents page 10 for page listing of the Shaft Side Load Capacity for each motor series).

Shaft Torque

$$\frac{T}{2\pi} = \frac{q\Delta P}{62.8} \text{ Nm} = \frac{\text{PSI} \times \text{in}^3/\text{rev}}{6.28} = \text{lb-in}$$

Shaft Speed

$$\text{RPM} = \frac{\text{Flow}}{\text{Displacement}} = \frac{1000 \times \text{l/min}}{\text{cm}^3/\text{rev}} \text{ RPM} = \frac{231 \times \text{GPM}}{\text{in}^3/\text{rev}}$$

Power (into motor)

$$\text{Kw} = \frac{\text{bar} \times \text{l/min}}{600} \text{ HP} = \frac{\text{PSI} \times \text{GPM}}{1714}$$

Power (out of motor)

$$\text{Kw} = \frac{\text{Nm} \times \text{RPM}}{9549} \text{ HP} = \frac{\text{lb-in} \times \text{RPM}}{63,025}$$

where: Kw = Kilowatt
 HP = Horsepower
 LPM = Liters per Minute
 GPM = Gallons per Minute
 Nm = Newton Meters
 lb-in = Pound inch
 Bar = 10 Newtons per Square Centimeter
 PSI = Pounds per Square Inch
 q = Displacement



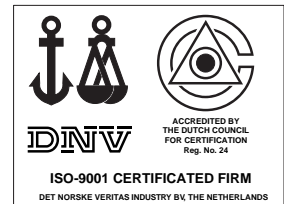
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